

Sub-kHz mechanical resonator passively cooled to 6 mK

Loek van Everdingen^{1,*}, Jaimy Plugge^{1,*}, Tim M. Fuchs^{1,2}, Guido L. van de Stolpe^{1,3}, Dalal Benali¹,
Thijmen de Jong¹, Jasper Bijl¹, Wim A. Bosch⁴, and Tjerk H. Oosterkamp^{1,†}

¹Leiden Institute of Physics, *Leiden University*, P.O. Box 9504, 2300 RA Leiden, The Netherlands

²School of Physics and Astronomy, *University of Southampton*, Southampton SO17 1BJ, United Kingdom

³E. L. Ginzton Laboratory, *Stanford University*, 348 Via Pueblo, Stanford, California, USA

⁴HDL, Hightech Development Leiden, Leiden, The Netherlands



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Highly coherent mechanical resonators are invaluable to ultrasensitive detection techniques by enabling detection of small forces. Studying mechanical resonators in a thermal equilibrium state at millikelvin temperatures provides a promising path to increase their coherence time. Here, we passively cool a 700 Hz massive (1.5 ng) mechanical cantilever down to 6.1(4) mK by means of nuclear demagnetization, as confirmed by detecting its thermal motion via a lock-in-based detection scheme. At the lowest temperatures the thermal motion of the resonator is still clearly distinguishable from the background noise. Our data analysis confirms that at these temperatures, the motion is still thermally distributed. These results pave the way for passive cooling low-frequency resonators to the sub-millikelvin regime, which would enable tests of quantum mechanics and advances in ultrasensitive force detection.

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Nanomechanical resonators play a vital role in fundamental physics, as they enable detection of small forces [1–3]. These forces play a role in topics such as nanoscale magnetic resonance imaging [4–7], solid-state physics [8–10], and small-scale measurements of gravity [11,12]. Additionally, large mass mechanical resonators play a role in studying the quantum-to-classical boundary by placing bounds on various effective wave function collapse models, such as Diosi-Penrose models and continuous spontaneous localization (CSL) [13–15]. Aforementioned experiments require mechanical resonators with favorable properties, such as high Q factors [16,17], low-force noise [2,18,19], and good displacement sensitivity [20,21]. These properties tend to improve at lower resonator temperature and consequently experiments are generally carried out at cryogenic temperatures [22].

To reach such low temperatures, both active and passive forms of cooling are explored. Active cooling requires continuous driving of a mode, such as feedback cooling [23] or sideband cooling [24], and has been employed to cool mechanical resonators in a wide range of frequencies into their ground state [17,25–27]. However, it is undesirable in experiments probing quantum systems, as it disturbs a system from mechanical equilibrium via the external drive. On the contrary, passive cooling leaves a mechanical system in thermal

equilibrium, once cold. It was first used to cool gigahertz systems to their ground state at the operation temperature of a dilution refrigerator [28]. A widely applied form of passive cooling is nuclear demagnetization, which has been used to cool microelectronics below 1 mK [29,30], and was recently employed to cool a 15 MHz device into its motional ground state [31].

However, bringing sub-kHz mechanical probes to sub-mK temperatures while leaving them in thermal equilibrium remains a formidable challenge [32]. If achieved, it has the potential to improve the Q factor of highly coherent mechanical resonators by mitigating mechanical loss channels [33,34]. Mechanical sensors with a high Q factor and low-force noise find application in the aforementioned experiments in ultrasensitive detection, which would consequently benefit from a decrease in temperature.

In this work, we apply nuclear demagnetization to cool a 1.5 ng mechanical resonator to 6.1(4) mK. By application of a lock-in detection scheme that tracks the resonator energy, we are able to verify the thermal equilibrium nature of its state through the direct observation of Boltzmann energy statistics. To our knowledge, this is the first observation of the equilibrium motion of a massive mechanical resonator in the hertz to kilohertz regime below the 20 mK base temperature of conventional dilution refrigerators.

Experimental setup. The experimental setup used in this work was originally designed for magnetic resonance force microscopy at low temperatures (see Fig. 1). A soft silicon cantilever [35] with a magnetic tip ($\text{Nd}_2\text{Fe}_{14}\text{B}$ sphere, 7.3 μm diameter) is suspended above a pickup loop in a pulse tube cryogen-free dilution refrigerator (Leiden Cryogenics CF-1200). To minimize mechanical vibrations, the experiment is mounted on a mass-spring suspension system hanging

*These authors contributed equally to this work.

†Contact author: oosterkamp@physics.leidenuniv.nl

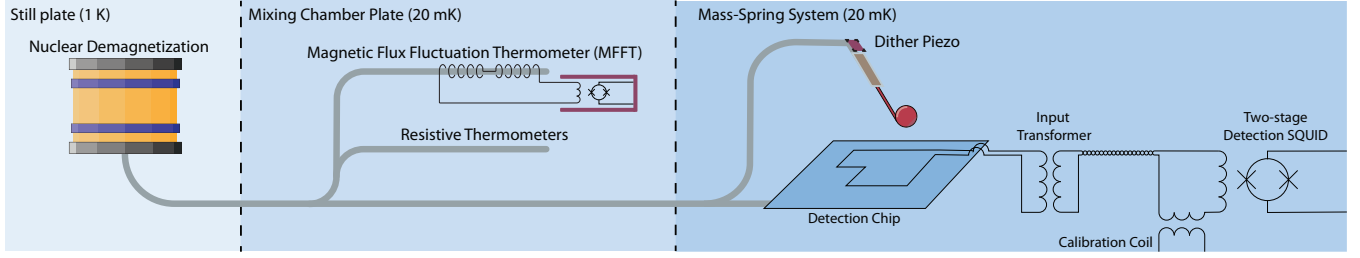


FIG. 1. Schematic illustration of the setup used for this work. A thermally isolated silver wire is linked to the nuclear demagnetization stage and connects different parts of the experiment, notably the cantilever, detection chip, and input transformer, to a temperature below the base temperature of the dilution refrigerator at 20 mK. The lower right corner of the schematic shows the circuit that is used for the detection and calibration of the cantilever motion. The position inside the dilution refrigerator of different parts is indicated. In the center part, the magnetic flux fluctuation thermometer (MFFT) is illustrated inside its lead shielding (purple). The rightmost part shows the cantilever suspended above the detection circuit.

inside the dilution refrigerator. The cantilever exhibits a resonance frequency $\omega_0 \approx 2\pi \times 700$ Hz and a spring constant of $26 \mu\text{N/m}$. Magnetic flux induced in the pickup loop generates a current in a superconducting quantum interference (SQUID) input coil of a two-stage readout SQUID (Magnicon NC-1 integrated two-stage current sensor), and hence a voltage signal. Precise positioning of the cantilever is achieved using a piezomotor system comprising three independently controllable spindles actuated by slip-stick piezomotors. Oscillations of the cantilever can be driven electrically through a dither piezo that excites the cantilever base. The amplitude of the cantilever oscillations can be calibrated through a calibration coil that excites the cantilever magnetically and is located between the pickup loop and the readout SQUID input coil. The cantilever frequency varies at different positions due to local forces acting on the cantilever. The local forces cause a change in the effective stiffness k_{eff} , which causes variations in the effective resonance frequency ω_{eff} .

To achieve cooling of the cantilever below the base temperature of the dilution refrigerator (~ 20 mK), we thermally anchor both the cantilever and the detection chip to a PrNi_5 nuclear demagnetization stage using a silver wire (contrary to previous work where only the cantilever was connected [34]). Crucially, the silver strip is routed along the mass-spring system using thermally isolating clamps, so that it is mechanically, but not thermally, connected to each mass. This allows the wire to reach sub-millikelvin temperatures (the mass-spring system is thermalized at ~ 20 mK), while retaining optimal vibration isolation [34]. Thermal isolation between the detection chip and the sample holder is achieved by placing the detection chip on top of a machined Macor® plate.

In order to accurately determine the temperature of the environment, three thermometers are connected to the silver wire, which forms a thermal link to the cantilever. We use

two resistance thermometers for temperatures between 15 and 250 mK (from Hightech Development Leiden, HDL), and a magnetic flux fluctuation thermometer for lower temperatures, which induces minimal heat dissipation [36]. The MFFT consists of a gradiometric coil wound around the silver wire that picks up magnetic fluctuations induced by Johnson-Nyquist noise, which is measured using a second Magnicon SQUID [34]. Details on the thermometer calibration procedure can be found in the Supplemental Material [37].

We directly infer the cantilever temperature by detecting its thermal motion through the ac magnetic flux signal induced in the SQUID pickup loop [38]. This measurement of the cantilever motion is minimally invasive, with added dissipation in principle only limited by the coupling between the cantilever and the detector. We measure the flux-to-voltage conversion parameter κ by driving the cantilever with an oscillating test flux generated by a calibration coil (see Fig. 1) and measuring the resulting flux through the pickup coil [34]. Note that here we only consider magnetic interactions, neglecting possible electrostatic contributions (e.g., due to residual charges on the cantilever tip). A detailed discussion on the calibration procedure can be found in the Supplemental Material [37].

This work presents experiments in two different cooldown cycles of the dilution refrigerator, for which the respective measurement parameters are presented in Table I. In each cycle, we start by varying the coupling between the detection circuit and the cantilever before turning on the nuclear demagnetization. The resulting conversion parameters κ are given in Appendix A: *Combined MFFT and cantilever temperature data*. In both cooldown cycles, the cantilever is more than $10 \mu\text{m}$ away from the surface of the detection chip while measuring.

Determination of the cantilever temperature through its thermal motion. To extract the cantilever temperature, we

TABLE I. Parameters of the two different measurement runs. T_{final} is the lowest temperature measured through the cantilever thermal motion during a run and T_{MFFT} is the lowest temperature measured through the MFFT.

Run	Frequency (Hz)	Q	κ (mV)	T_{final} (mK)	T_{MFFT} (mK)
A	746.6	13 200 (100)	$9.6(6) \times 10^{-6}$	6.1(4)	3.4(4)
B	669.7	15 400 (400)	$1.9(2) \times 10^{-5}$	7.7(4)	3.1(2)

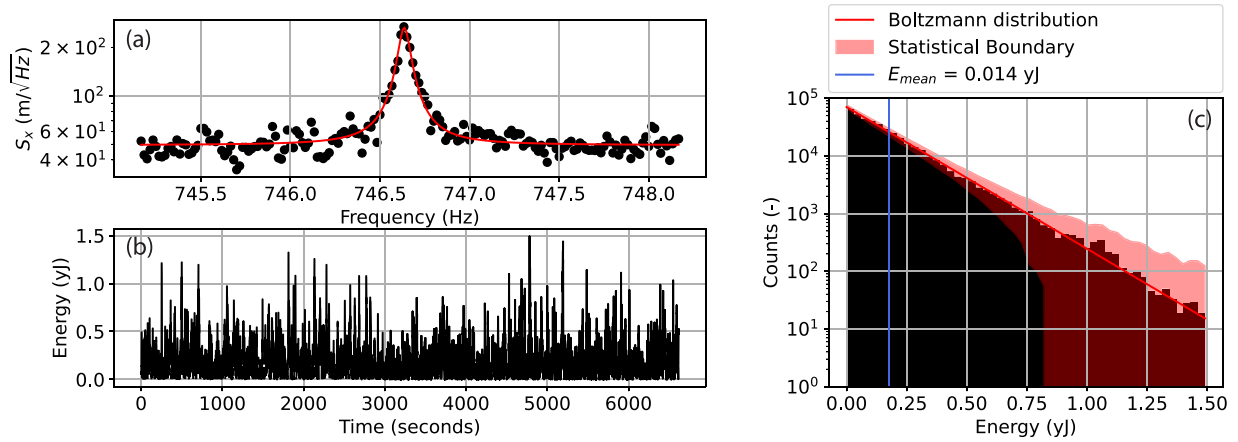


FIG. 2. The cantilever temperature is determined from the readout SQUID signal after postprocessing. (a) The resonance frequency of the cantilever is determined through a frequency sweep driven by the piezo. This is done through a Lorentzian fit over the range where the peak stands out above the detector noise. (b) The (undriven) thermal motion of the cantilever. A digital lock-in amplifier is used at the determined resonance frequency on the SQUID time signal to obtain the cantilever amplitude as a function of time. (c) After plotting the energy in a histogram, the cantilever temperature is obtained from the mean energy and cross-checked through the slope of the energy distribution. The red shaded area indicates one standard deviation in the number of counts per bin.

apply a lock-in-based detection scheme that enables real-time tracking of the cantilever energy (here executed in postprocessing; see the Supplemental Material [37] for details on the analysis). This type of analysis was first implemented by Golokolenov *et al.* [39]. Such an energy time trace facilitates immediate insight into the resonator thermodynamics [see Fig. 2(b)], which we use here to verify the thermally limited nature of its motion [Fig. 2(c)]. That is to say, it allows to check for occasional large mechanical disturbances that are not filtered by the mass-spring system. This method offers a clear and intuitive way to analyze the cantilever signal, and can be considered complementary to frequency-domain methods [34].

The thermal motion of the cantilever is given by $\langle x \rangle^2 = k_B T/k$, where $\langle x \rangle^2$ is the time-averaged amplitude of the cantilever displacement, k_B is the Boltzmann constant, k is the cantilever stiffness (by invoking the equipartition theorem [40]). The cantilever frequency can vary due to local forces, such as Meissner repulsion induced by the superconducting structures of the readout circuit or due to magnetic interactions with surface spins [34,41]. We measure the effective resonance frequency ω_{eff} by exciting the cantilever magnetically through the calibration coil or mechanically through the piezo and fitting a Lorentzian to the Fourier transform of the SQUID signal [10 min of integrated signal is shown in Fig. 2(a)].

The (digital) lock-in amplifier is applied to the measured time signal from the readout SQUID at resonance frequency ω_{eff} . We set the lock-in bandwidth to 1 Hz, approximately two times the bandwidth over which the thermal motion of the resonator stands out above the detection noise [see Fig. 2(a)]. The correlation time of the cantilever is $\tau = 2\frac{Q}{\omega_0}$, which determines the timescale at which its energy decays. A value for τ can be determined by sweeping cantilever excitations around the resonance frequency. Typically, we find $\tau \approx 7$ s. Hence, 2 h of data, as presented in Fig. 2(b), contain about 1028 independent measurements of the cantilever energy, from which we infer the cantilever temperature by taking the mean. To

verify the thermal nature of the data, we compare a histogram to the Boltzmann distribution, which scales as $\sim \exp(\frac{E}{-k_B T})$ [Fig. 2(c)]. A cutoff is visible (i.e., the number of counts drops abruptly from 25 to 0) at high energy, because the sample rate of the lock-in is much faster than the rate at which the cantilever energy varies, determined by τ . External (mechanical) excitations of the cantilever motion would cause the histogram in Fig. 2(c) to deviate from a Boltzmann distribution.

To cross-check the temperature, we define an expression for the standard deviation δn of the distribution. Given the number of independent measurements in a bin is equal to t_{bin}/τ , with t_{bin} the bin duration, we expect the relative variation to be

$$\frac{\delta n}{n_{\text{bin}}} = \frac{1}{\sqrt{t_{\text{bin}}/\tau}}, \quad (1)$$

for a bin that contains n_{bin} counts. Ninety-five percent of the data are expected to fall within an interval of $\pm 2 \delta n$ around Boltzmann distribution, which is confirmed by our analysis [see Fig. 2(c)]. By fitting the slope of the distribution, we extract a cantilever temperature of 10.3(2) mK (where the value in parentheses denotes the uncertainty on the last digit).

Thermal motion below 10 mK. The results in this work concern a comparison between two independent datasets of the cantilever temperature. These datasets are used to discuss the factors that cause saturation of the cantilever temperature and factors that cause offsets in the calibration procedure for the cantilever amplitude.

Each dataset consists of a measurement of thermal fluctuations of the cantilever when changing the current through the nuclear demagnetization coil. The magnetic field of the nuclear demagnetization coil is first ramped up to 2 T. The energy released during this process is thermalized to the dilution refrigerator through an aluminum heat conductance switch. After the field has reached 2 T and the demagnetization stage has thermalized, the heat conductance switch is flipped to minimize thermal conductivity. Then, the field is reduced

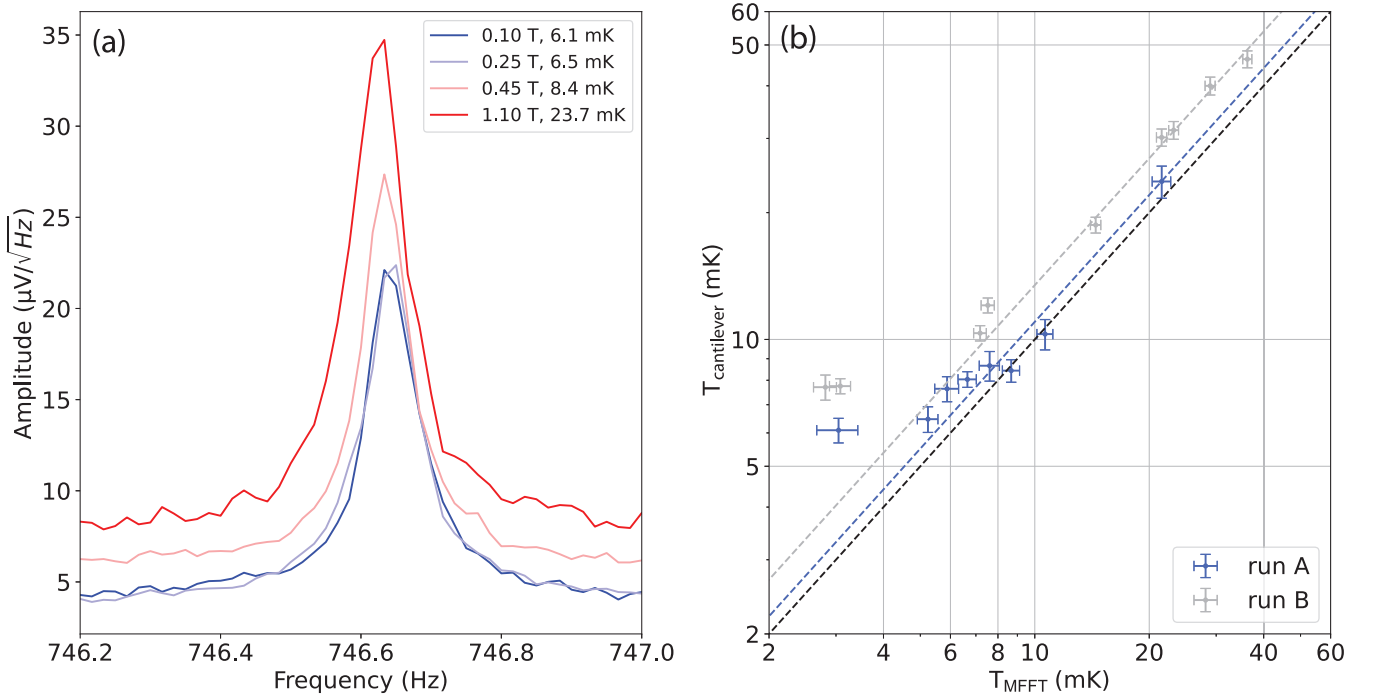


FIG. 3. Temperature determined from the cantilever thermal motion vs bath temperature. Panel (a) shows the thermal motion as observed in the power spectral density during measurement run A at various magnetic fields in the nuclear demagnetization stage. In panel (b), the cantilever temperature $T_{\text{cantilever}}$ is plotted against the MFFT temperature T_{MFFT} during run A and run B. The dashed lines indicate $T_{\text{cantilever}} = cT_{\text{MFFT}}$ corresponding to run A and run B. The black dashed line indicates $c = 1$.

stepwise, providing cooling power to the silver wire through the demagnetization of the PrNi_5 . At the lowest magnetic field, the cantilever and detection chip are allowed to thermalize for at least 12 h, while the amplitude of the cantilever resonance is being monitored and the MFFT acquires temperature data. When no further decrease in the measured cantilever power spectral density (PSD) is observed, the current through the nuclear demagnetization coil is increased stepwise, while monitoring the cantilever amplitude. After each step, we wait for the MFFT temperature to reach a stable value after which 2 h of data are acquired.

The thermal motion measured in dataset A is plotted as a PSD in Fig. 3(a) to be able to observe a decrease in amplitude at lower temperatures. The final temperature is extracted from the corresponding histograms, which can be found in *Appendix B: Thermal distributions of cantilever motion*. During all measurements, the thermal fluctuations of the cantilever can clearly be distinguished from the background detection noise. The measured thermal fluctuations of the cantilever, plotted as a cantilever temperature $T_{\text{cantilever}}$, are plotted against the MFFT temperature T_{MFFT} in Fig. 3(b). Horizontal error bars indicate the standard deviation of the temperature measurements from different spectra throughout this time interval. If the cantilever is in thermal equilibrium, its temperature is expected to fluctuate around an average value $\bar{T}_{\text{avg}}$. The standard deviation of these fluctuations over a time interval t_{meas} is described using $\Delta T = \sqrt{\tau/t_{\text{meas}}} \bar{T}_{\text{avg}}$ around the average temperature.

Throughout both datasets, we observe that $T_{\text{cantilever}} > T_{\text{MFFT}}$ to within the uncertainty of $T_{\text{cantilever}}$. At the lowest temperatures, $T_{\text{cantilever}}$ is saturated and does not decrease further,

at a saturation temperature of 6.1(4) and 7.7(4) mK for run A and run B, respectively. Even at the lowest temperatures, the cantilever motion is still following a Boltzmann distribution, hinting at a thermally distributed origin of the saturation.

To quantify the deviation between $T_{\text{cantilever}}$ and T_{MFFT} , we fit $T_{\text{cantilever}} = cT_{\text{MFFT}}$ for the data with $T_{\text{MFFT}} > 8$ mK. The uncertainty in the fit parameter is estimated from the square root of its variance. However, the uncertainty is dominated by the limited amount of fitting points, which is not reflected in this value. For the data in run A, the value for $c = 1.07(3)$, close to 1, which is expected when the cantilever and MFFT are well thermalized to each other. However, in run B, c is equal to 1.33(3). As no changes were made to the silver wire or cantilever between datasets A and B, we suspect the deviation in c to be the result of inaccuracies in the displacement calibration of the cantilever. A possible origin is the presence of electrostatic forces driving the cantilever. The calibration only takes into account forces due to generated magnetic fields. However, if there are any residual charges on the cantilever tip, electrostatic forces can act as an additional drive of unknown sign on the cantilever due to parasitic capacitances. *Appendix C: Offset in the cantilever temperature conversion* discusses potential inaccuracies in the displacement calibration in more depth. If the calibration in run B was lower such that $c = 1$, it would result in a saturation temperature around 6 mK, similar to dataset A.

We deem it unlikely that the saturation of the cantilever temperature around 6 mK is due to external vibrations. Vibrations would result in day-night variations of the cantilever temperature. We do not observe such variations, as is visible in *Appendix A: Combined MFFT and cantilever temperature*

data. We fit the function $T_{\text{cantilever}} = (T_{\text{MFFT}}^n + T_0^n)^{1/n}$ to the data of run A. The free parameters are the saturation temperature T_0 and coefficient n . The value of n can be used to determine the limiting thermal resistance [38,42]. This yields $n = 4(2)$ and $T_0 = 6(1)$ mK. This is consistent with a thermal resistance coupled in through the detection chip. A hypothesis for these saturations is heating due to the thermal dissipation of the dc bias voltage in the SQUID detection circuit.

Discussion. Through the use of nuclear demagnetization in combination with a vibration isolation system, we demonstrated passive cooling of a nanomechanical cantilever to temperatures below 10 mK, measuring its equilibrium motion at temperatures down to 6.1(4) mK. This result paves the way for improvements in the detection sensitivity of nanomechanical sensors by improving their force noise and Q factor through equilibrium cooling. Additionally, it constitutes a step toward future tests of wave function collapse models such as CSL [13].

The success of these experiments is dependent on the force noise of the cantilever for which we can make an estimate using the method of Stowe *et al.* [22]. Our current cantilever has a resonance frequency of approximately $2\pi \times 700$ Hz, effective mass is $m_{\text{eff}} \approx 1.5$ ng, and $Q \approx 14\,000$. At 6.1 mK, this yields $\sqrt{S_F} \approx 3.9 \times 10^{-19}$ N/ $\sqrt{\text{Hz}}$. This is on the same order as the best efforts to optimize force noise in the low-kHz regime [43,44] and 2 orders of magnitude higher than the state of the art for clamped mechanical resonators [1,21].

This figure can be improved by a number of changes to the experiment. The experiments performed by van Heck *et al.* were carried out with the exact same cantilever, which then had $Q \approx 40\,000$. It is unclear what caused the deterioration of its Q factor. A higher Q factor can be achieved by switching to nanoladder cantilevers [2]. A nuclear demagnetization stage using PrNi₅ can theoretically reach a minimal temperature of 0.5 mK. By combining $Q \approx 40\,000$ with a cantilever temperature of 0.5 mK, we can potentially achieve 6.8×10^{-20} N/ $\sqrt{\text{Hz}}$.

There are two issues to overcome in order to cool a cantilever to 0.5 mK. First, one has to remove any mechanisms that currently cause saturation of the cantilever temperature. Potential origins are heating caused by the SQUID detection circuit and free electron spins on the detection chip that are excited by spurious magnetic field fluctuations. Second, one can improve the techniques to lower the demagnetization temperature by either increasing the cooling power of the experiment or reducing the heat input through conduction or mechanical vibrations. This is possible through a second nuclear demagnetization system, utilizing nuclear spins in copper, that is precooled using the existing nuclear demagnetization coil.

Finally, for future experiments, it is important to reduce uncertainty in the calibration procedure. This requires a way to mitigate the effect of parasitic capacitances throughout the magnetic calibration or by reducing any remaining charges on the cantilever, making it less susceptible to the electrostatic driving force.

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Data availability. The data that support the findings of this Letter are not publicly available upon publication because it is not technically feasible and/or the cost of preparing, depositing, and hosting the data would be prohibitive within the terms of this research project. The data are available from the authors upon reasonable request.

Appendix A: Combined MFFT and cantilever temperature data. The combined temperature measurement through the MFFT and the temperature measured from the cantilever motion are displayed in Fig. 4. Table II contains the parameters resulting from the displacement calibration (details are outlined in the Supplemental Material [37]).

In the MFFT data presented in Fig. 3, a data point is added for the temperature obtained from the spectrum from

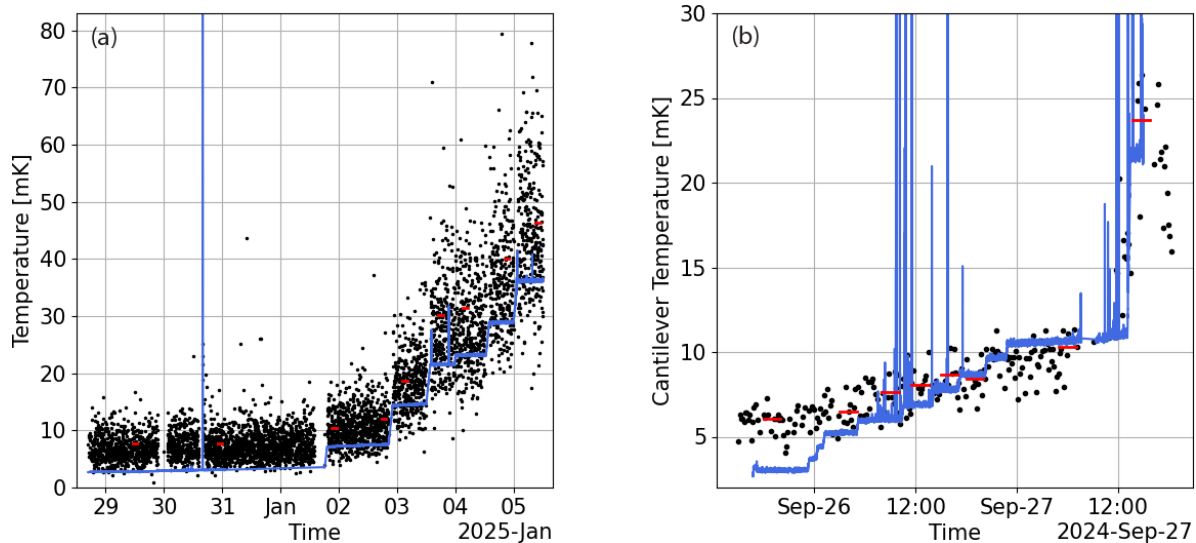


FIG. 4. Temperature steps measured during run A [panel (a)] and B [panel (b)]. The temperature of the MFFT is plotted in blue and the cantilever temperature in black. Red bars indicate the time segments used to calculate the data points in Fig. 3(b).

TABLE II. Parameters for the cantilever displacement calibration as measured for the different runs in this work. Sample rate refers to the sample rate of the detection SQUID. The parameters β and κ are, respectively, the energy coupling and conversion parameter from the displacement calibration. The temperatures T_{final} and T_{MFFT} are the lowest temperatures measured from the cantilever motion and through the MFFT, respectively.

Run	Sample rate (Sa/s)	Frequency (Hz)	Q	β	$Q\beta^2$	κ (m/V)	T_{final} (mK)	T_{MFFT} (mK)
A	200 000	746.6	13 200 (120)	$1.8(6) \times 10^{-3}$	$4.4(6) \times 10^{-2}$	$9.6(6) \times 10^{-6}$	6.1(4)	3.4(4)
B	50 000	669.7	15 400 (400)	$1.0(5) \times 10^{-3}$	$1.6(4) \times 10^{-2}$	$1.9(2) \times 10^{-5}$	7.7(4)	3.1(2)

all datafiles (respectively, 16 and 61 s of measurement during run A and run B) obtained during a run. The spikes in the temperature occur when a change is made to a dc offset voltage in the electronics of the MFFT SQUID. This does not change the IV curve of the SQUID and resulting sensitivity for changes in magnetic flux. However, due to the jump in the output voltage, a spike appears in the temperature data after processing.

Appendix B: Thermal distributions of cantilever motion. In Fig. 5, the histograms of the cantilever energy corresponding to the curves in Fig. 3(a) are plotted. As indicated in the results section, variations in the cantilever energy occur over an interval determined by the inverse cantilever time constant $1/\tau$. This is much slower than the sample rate of the digital lock-in amplifier, which is 100 Sa/s. As a result, bins with less than $\tau/10 \times 100 \text{ Sa/s} \approx 70$ counts have little statistical significance.

Appendix C: Offset in the cantilever temperature conversion. To obtain a quantified measurement of the cantilever temperature, several data processing steps are relevant. Here,

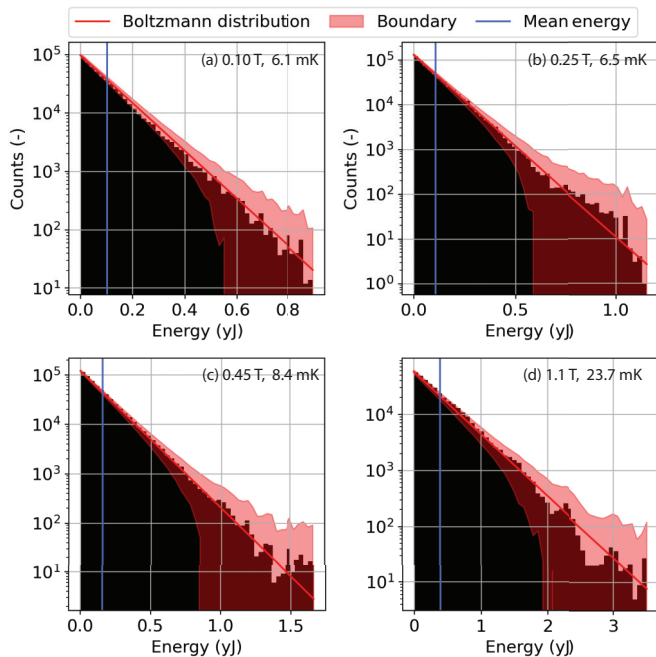


FIG. 5. The histograms of the cantilever motion after applying a digital lock-in amplifier of the curves in Fig. 3(a). The red line marks the expected number of counts in each bin if they are drawn from a Boltzmann distribution with a width determined by the temperature as calculated from the mean energy. The red shaded area marks one standard deviation around this distribution.

we discuss several issues that we identified in this procedure. Any inaccuracy can result in both a too low or too high figure for the cantilever temperature. We first present a potential systematic error due to deviations in the mass of the cantilever tip. Then, we discuss the effect of electrostatic driving of the cantilever during the displacement calibration.

From the measured amplitude of the cantilever oscillations, the cantilever energy is calculated using a proportionality constant $E \propto m_{\text{eff}}\omega^2$. The energy thus scales linearly with the effective cantilever mass, which is dominated by the mass of the spherical magnetic tip. We now discuss the uncertainty in the mass of this magnetic tip. The tip diameter was observed to be $7.3 \mu\text{m}$, using a scanning electron microscope. Due to variations between different measurements, we estimate that this value is precise to about 8%. Using the density of $\text{Nd}_2\text{Fe}_{14}\text{B}$ ($\rho = 7450 \text{ kg/m}^3$), we find that this translates to a mass of the tip of $1.51(35) \text{ ng}$. The cantilever mass is around 0.12 ng . Taking into account the uncertainty in the mass of the tip, the uncertainty in the effective mass is found to be on the order of 23%. This directly translates to a systematic uncertainty in the cantilever temperature, as $T_{\text{cantilever}} \propto m$.

We observed that it was possible to electrostatically drive the cantilever through the calibration coil. Figure 6 shows a sweep through the calibration coil, during which the output of the lock-in amplifier was grounded on two sides. This means that no current can run through the calibration coil and any effect is due to voltage changes. However, the cantilever is still resonantly driven during this sweep. This indicates that

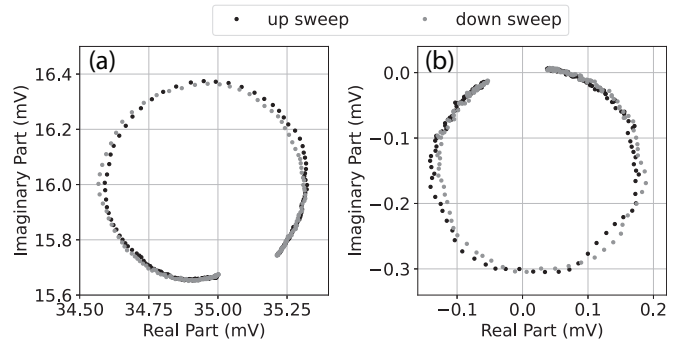


FIG. 6. Two frequency sweeps through the calibration coil. In panel (a), the sweep is conducted in the normal configuration, with the input of the calibration coil connected to the output of the current source. In panel (b), the sweep is conducted to only allow for electrostatic driving of the cantilever. In both panels, a circle is visible that the cantilever is excited resonantly. However, in panel (b), there is no net offset of the circle from the origin, as there is no magnetic flux going through the calibration coil. Additionally, the circle is rotated with respect to the sweep conducted in panel (a).

the cantilever tip contains a nonzero electric charge, which is possibly acquired due to a touch of the detection chip while positioning. Whether electrostatic driving happens in parallel to inductive driving, the measured energy coupling β^2 can deviate from purely inductive driving. If the resulting conversion parameter α is too high or too low depends on the

phase difference with which both driving mechanisms act on the cantilever. It was not possible to calculate the coupling of this effect using the procedure outlined in the Supplemental Material [37], as no offset flux is present during the SQUID measurement. This suggests there the SQUID signal is not affected through inductive driving during this measurement.

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